

# Determination of Stresses on the Carbide Insert While Grooving By Using Ansys

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**Abstract**— This paper presents, the determination of stresses acting on a carbide insert while performing a grooving operation will investigate experimentally. In grooving operation, different types of stresses are developed and due to those stresses, various wear are produced on insert tips such as crater wear, flank wear, thermal wear and mechanical wear. So that to overcome the effect of wear, we are performing the grooving operation at different working conditions to increase the efficiency, surface finishing and life of the insert. Stresses developed on insert tip are analyzed theoretically by orthogonal formulation and experimentally by using finite element analysis software ANSYS. Actual values of stresses acting on tip of the insert and calculated theoretical values of stresses then compared at different parameters.

**Keywords:** Grooving, CVD, Wear, Forces, Analysis

## I. INTRODUCTION

Grooving is a multistep machining operation. It is a process of forming a narrow cavity of a certain depth, on a cylinder, cone, or face of a part. The groove will be in the shape of the cutting tool. For single-cut grooving generally, straight cuts can be made for groove widths of up to 8 mm giving the best method, chip control, and tool-life. The most common methods of rough machining wide grooves, or turning between shoulders, are multiple grooving, plunge turning, and ramping [2].

Recommended ring width is 0.6 to 0.8 times the insert width. This is a flexible method that is quick to program and geometry GM is the first choice for this method. To achieve the best roughing results in the form of a flat groove-bottom with the best groove-side quality. To achieve the best finishing results, care should be taken when machining the corners of the groove Then optimize the process in relation to the batch sizes. The Wiper effect generates a good surface finish with Ra values down to 0.2 microns [2].

## II. GROOVING

Machining grooves have many similarities to parting off, especially deep grooves. Although the same tool holder systems can be used for both parting and grooving in many instances, the insert geometries are dedicated to providing optimum performance and results. There are shallow grooves, deep grooves, wide grooves, external grooves, internal grooves, and face grooves. The insert corner is protected and chips are directed into the middle of the chip breaker [2].

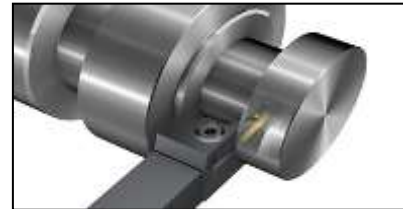


Fig. 1: Grooving

### A. Types of Grooving

- 1) Circlip grooving: The need of circlips on shaft and axle components is very common and there are two systems suitable for these grooving operations, and also have specific widths for circlip grooves.
- 2) Face grooving: A face grooving operation is carried out for remove the material from the face of workpiece.
- 3) Internal grooving:- Most of the methods for external grooving can be applied to internal grooving. Precautions may have to be considered, to ensure chip evacuation and to minimize vibration tendencies.

### B. Inserts

An insert is a tool which uses for the grooving operation, the insert will work at a relatively high cutting speed and must be able to resist deformation. Therefore, needs more tough tool material for this operation. Advanced insert geometries are necessary for performing parting and grooving operations in a satisfactory way [2].

For this project, the selected material for deep grooving operation is carbide which is CVD coated type carbide insert. Coating of TiCN and TiN is done on carbide insert and improved toughness and stability due to specialized carbide substrate with plastic deformation resistance [6].

SN	Quantity	Value	Unit
1	young's Modulus	530000-700000	MPa
2	Transverse strength	2.7	GPa
3	Fatigue	270-300	MPa
4	Hardness	13.7	Gpa
5	Ratio	0.31	-

Table 1: Mechanical Properties

A positive-rake insert gives lower cutting forces and thus lower pressure on the workpiece and this will reduce the size of the tip. On neutral or straight inserts the lead angle is zero. A reduced lead angle produces larger radial cutting forces but this can cause vibration problems especially when small diameters are machined [5].

With grooving and parting operations, the insert has machined surfaces on both sides of the feed direction.



$$= 164.80 \text{ m/min} \quad \dots 7$$

**E. Working Conditions**

$$\text{Spindle power} = 27 \times 10^3 \text{ W}$$

$$\text{Spindle speed} = 1000 \text{ RPM}$$

We know that

$$P = \frac{2\pi NT}{60} \quad T = \frac{27000 \times 60}{2\pi \times 1000} \quad \dots 8$$

$$T = 257.83 \text{ N-m}$$

Torque = Force × perpendicular distance

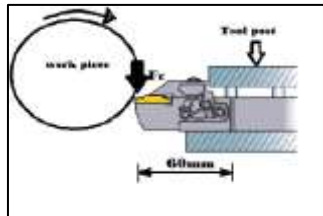


Fig. 5: Position of cutting force

$$\text{Perpendicular distance from tool post} = 0.06 \text{ m}$$

$$\text{Force} = 257.83 \div 0.06 = 4297.18 \text{ N}$$

**F. Forces in Metal Cutting**

Metal cutting involves a large amount of force. The power supplied for cutting is utilized mainly in shearing the material and overcoming the frictional forces between the tool and the chip. It is important to ascertain the cutting forces acting during metal cutting, for calculation of power required in metal cutting to properly design cutting tools, for a proper design of jigs and fixtures used to hold the workpiece during the cutting operation, and for selection of suitable operating conditions like speed, feed, and depth of cut.

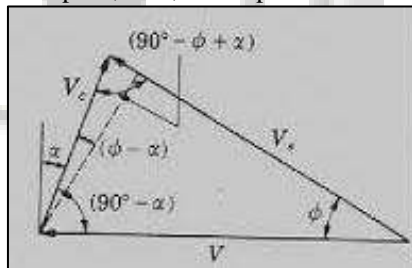


Fig. velocity triangle

Cutting force

$$F_c = 4297.18 \text{ N} \quad \dots 9$$

$$F_c = R \cdot \cos(\beta - \alpha)$$

$$4297.18 = R \cdot \cos(32.21 - 7)$$

Resultant force

$$R = 4749.56 \text{ N} \quad \dots 10$$

Tangential force

$$F_t = R \cdot \sin(\beta - \alpha) = 4749.56 \sin(32.21 - 7) = 2023.01 \text{ N} \quad \dots 11$$

Fiction force

$$F = R \cdot \sin(\beta) = 4749.56 \sin(32.21) = 2531.62 \text{ N} \quad \dots 12$$

Normal to Fiction force

$$N = R \cdot \cos(\beta) = 4742.56 \cos(32.21) = 4018.60 \text{ N} \quad \dots 13$$

Shear force

$$F_s = F_c \cdot \cos(\phi) - F_t \cdot \sin(\phi)$$

$$= 4297.18 \cos(32.395) - 2023.01 \sin(32.395) = 2544.60 \text{ N} \quad \dots 14$$

Normal to shear force

$$F_n = F_c \cdot \sin(\phi) + F_t \cdot \cos(\phi) = 4297.18 \sin(32.395) + 2023.01 \cos(32.395) = 4010.60 \text{ N} \quad \dots 15$$

Stress calculations

Here,

Width of cut (w) = 5mm

Uncut chip thickness (t<sub>0</sub>) = 0.2mm

Cross sectional area of the chip before removal (A<sub>o</sub>) = 1mm<sup>2</sup>

Area of shear plane (A<sub>s</sub>)

$$A_s = \frac{A_o}{\sin \phi} = \frac{1}{\sin 32.395} = 1.8665$$

$$\text{Shear stress } \tau = \frac{F_s}{A_s}$$

From the above formula values of stresses at different RPM are as below

SN.	N (RPM)	Forces (N)	τ (shear stress) Mpa
1	1000	4378.18	1363.29
2	1400	3069.41	973.78
3	1800	2387.32	757.24
4	2200	1953.16	619.94

**V. MODELLING AND ANALYSIS**

**A. Modelling**

After determining the type of analysis required and the characteristics of the operating environment, the engineer must produce a finite element model with appropriate analysis parameters, such as loads, constraints, and a suitable mesh.

Modeling of grooving insert is carried out in CAD software Creo Parametric 2.0. We drafted the design in the sketcher module with the available dimensions of the blade. Modeling was finalized in part design workbench.

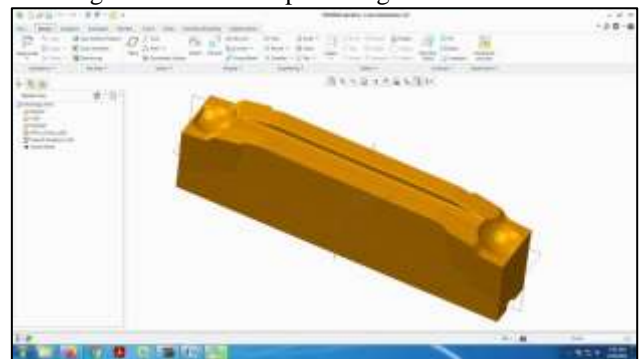


Fig. 6: grooving insert in part modeling in Creo Parametric

**B. Analysis**

**1) Introduction to ANSYS software**

ANSYS is a general-purpose finite-element modeling package for numerically solving a wide variety of mechanical problems. These problems include static dynamics, structural analysis (both linear and nonlinear), heat transfer, and fluid problems, as well as acoustic and electromagnetic problems. In general, a finite-element solution may be broken into the following three stages.

- a) Pre-processing: defining the problem  
 The major steps in pre-processing are
- 1) Define key points/lines/areas/volumes,
  - 2) Define element type and material/geometric properties, and
  - 3) Mesh lines/areas/ volumes as required.

The amount of detail required will depend on the dimensionality of the analysis, i.e., 1D, 2D, Axisymmetric, and 3D.

Solution: assigning loads, constraints, and solving  
 Here, it is necessary to specify the loads (point or pressure), constraints (translational and Rotational), and finally solve the resulting set of equations.

*C. Post-processing: further processing and viewing of the results*

- In this stage, one may wish to see
- 1) Lists of nodal displacements,
  - 2) Element forces and moments,
  - 3) Deflection plots, and
  - 4) Stress contour diagrams or temperature maps

Modeling and analysis of Grooving insert are presented. For Modeling of Grooving insert we used CAD software Creo Parametric 2.0. We drafted the design in the sketcher module with the available dimensions of the insert. Modeling was finalized in part design workbench. For analysis, we used analysis software ANSYS 19 R3. After preparing a solid Geometry model the important steps are meshing and applying loading and boundary conditions

**VI. RESULTS AND DISCUSSION**

The analytical results of the maximum cutting force acting on the insert ( $F_c$ ), torque ( $T$ ), were obtained also we obtained the force and torque on the grooving insert for three different RPM and Coefficient of friction. All these results are presented in this.

*A. Analytical results*

We have calculated cutting forces by an analytical method and they are 4297.18N,3069.41N,2387.32N,1953.16N at 1000 RPM, 1400 RPM, 1800 RPM, and 2200 RPM. On the other side, stresses that develop at insert are as follows.

Sr. No	RPM	Stress (MPa)
1	1000	1363
2	1400	973.78
3	1400	757.24
4	2200	644.78

Table 1: Theoretical stresses at different RPM

*B. Software results*

Modeling and analysis of grooving insert were carried out in CATIA and ANSYS software respectively. We obtained results for different RPM. For that, we use the force acting on the holder calculated in section and for three RPM. Results are shown below

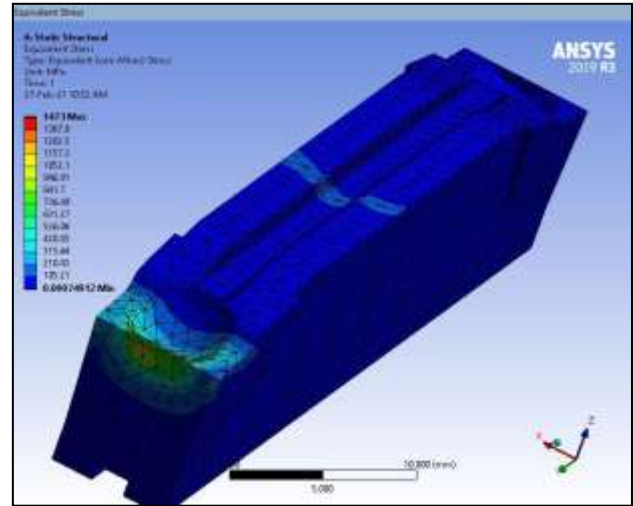


Fig. 7: Von misses stress on grooving insert in ANSYS

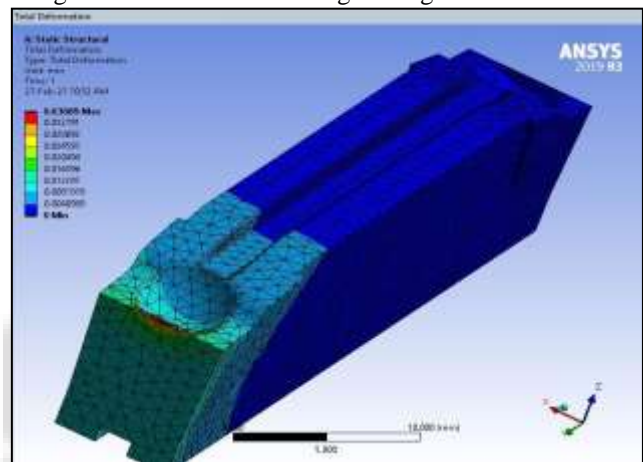


Fig. 8: Grooving insert nodal displacements in ANSYS

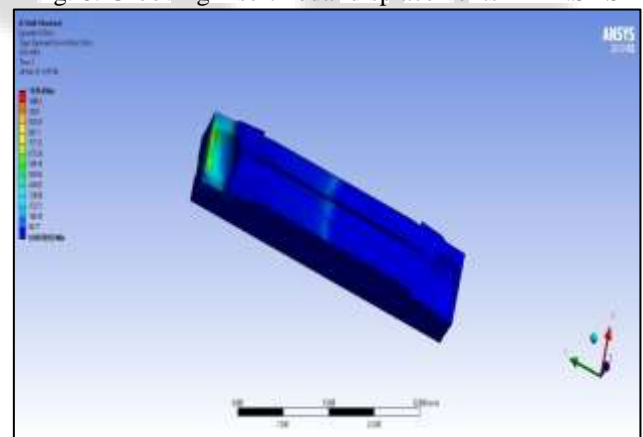


Fig. 9: Von misses stress on grooving insert in ANSYS

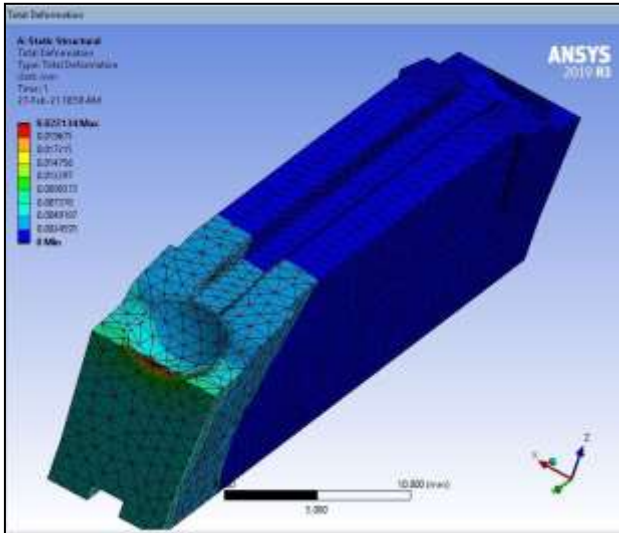


Fig. 10: Grooving insert nodal displacements in ANSYS

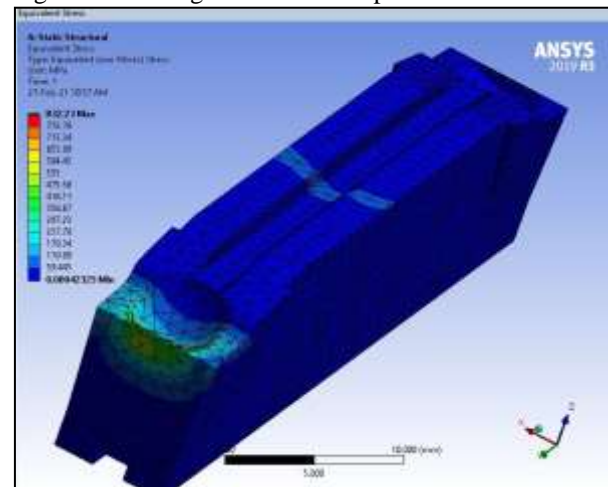


Fig. 11: Von mises stress on grooving insert in ANSYS

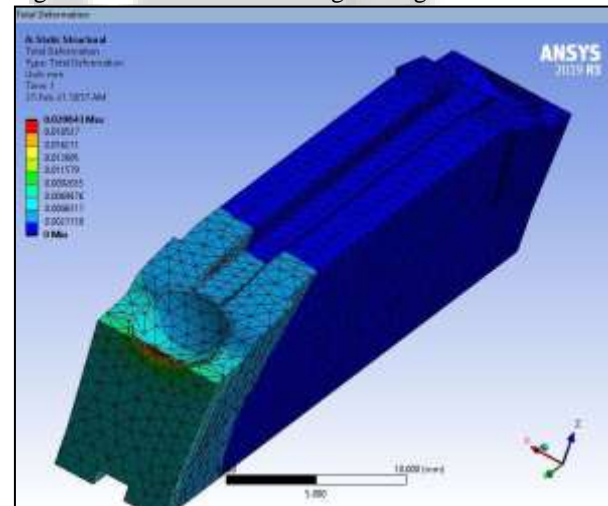


Fig. 12: Grooving insert nodal displacements in ANSYS

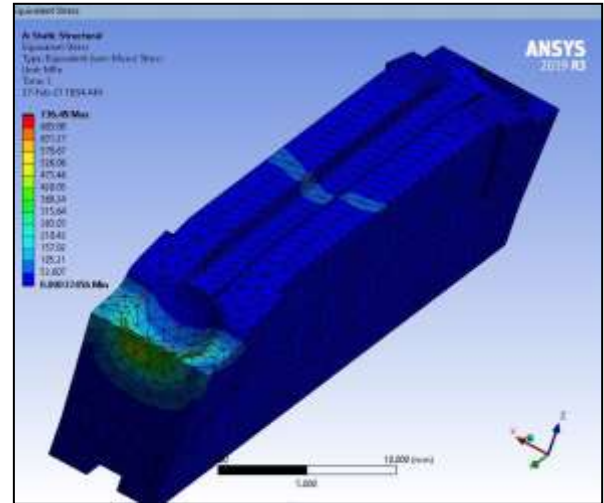


Fig. 13: Von mises stress on grooving insert in ANSYS

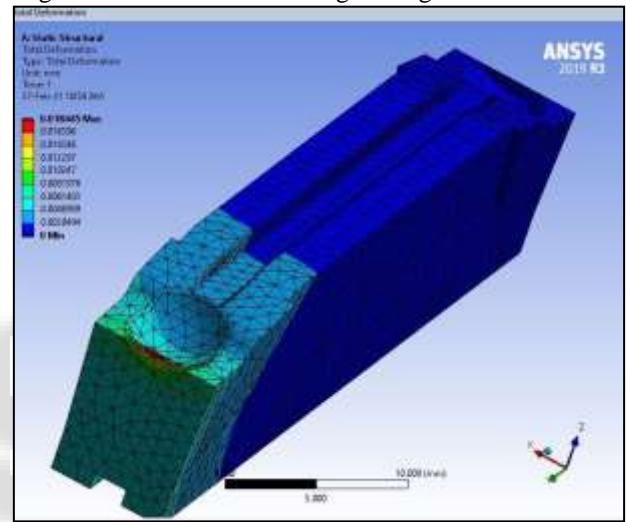


Fig. 14: Grooving insert nodal displacements in ANSYS

Sr. No	RPM	Von mises Stress (M Pa)
1	1000	1473
2	1400	1178.4
3	1400	832.32
4	2200	736.49

Table 2: Von mises stresses at different RPM

Results obtained from the calculations and the software are presented. The Analytical results of the maximum cutting force acting on the Grooving insert ( $F_c$ ), torque ( $T$ ), Von mises stress was presented in the software results (with diagrams) were presented in which includes vector displacement diagram and Von mises stress for three different RPM and Torques.

Sr. No	Theoretical stresses (Mpa)	Von mises Stress (MPa)
1	1363	1473
2	973.78	1178.4
3	757.24	832.32
4	644.78	736.49

Table 3: stresses Comparison

## VII. CONCLUSION

Here, by the overall study of grooving Insert and its failure pattern, we conclude the following points

- 1) CAD method has been explored for the design and analysis of Grooving Insert and Finite Element Analysis was done for investigation of stresses experienced by the Insert. A comparison was made between the stresses acting on the tip of the insert for different RPM of the CNC spindle.
- 2) The comparative study showed that stresses are minimum for the insert when grooving at 2200 RPM compared to Grooving at 1000 RPM. This ultimately shows the Grooving Inserts life is dependent upon torque and RPM.
- 3) Calculation of shear stress and von mises stress showed that at all the four values of N (RPM), values of yield strength are less than calculated values. So Insert is going to fail at this working condition.

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